Large-eddy simulation of lean hydrogen–methane turbulent premixed flames in the methane-dominated regime

Francisco E. Hernández-Pérez*, Clinton P.T. Groth, Ömer L. Gülder

University of Toronto Institute for Aerospace Studies, 4925 Dufferin Street, Toronto, Ontario M3H 5T6, Canada

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The application of large-eddy simulation (LES) to the prediction of H2-enriched lean methane–air turbulent premixed combustion is considered. A presumed conditional moment (PCM) subfilter-scale combustion model is coupled with the flame prolongation of intrinsic low-dimensional manifold (FPI) chemistry tabulation technique. The LES and PCM-FPI modelling procedures are then applied to the prediction of laboratory-scale axisymmetric Bunsen-type turbulent premixed flames. Both premixed methane–air and H2-enriched methane–air flames are considered and the predicted solutions are examined and compared to available experimental data. The enriched flame has 20% H2 in terms of mole fraction and lies in the methane-dominated regime of hydrogen–methane mixtures. The LES simulations predict similar qualitative trends to those found in the experiments for flame height and curvature. The addition of H2 decreases the flame height and broadens the curvature probability density functions, which show a Gaussian-type shape centred around zero. Moreover, the enriched flame displays a higher degree of wrinkling with sharper ridges of negative curvature and larger pockets of positive curvature. Overall, the proposed treatment for the PCM-FPI combustion model, in terms of progress variable and tabulated data, seems to perform well for the H2-enriched methane flame in the methane-dominated regime.

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1. Introduction

Despite the widely-recognized potential of H2 as a cleaner and sustainable source of energy, the approaches for the transition from a fossil-based to hydrogen-based economy are still under discussion and the implementation of the hydrogen-based economy is expected to take decades [1,2]. Muradov and Veziroğlu [2] have extensively reviewed such transition, highlighting carbon-neutral technologies and processes (see, e.g., Ref. [3]). H2 and hydrocarbon fuel blends appear to be a promising option to synergistically pave the way toward pure hydrogen-based combustion systems while alleviating green-house gas and pollutant emissions related to fossil fuel combustion. The possibility of using hydrogen-enriched hydrocarbon fuels as a
means for enabling greater stability of lean premixed flames with significantly reduced emissions of nitrogen oxides is also very appealing.

While promising, the widespread application of hydrogen-enriched hydrocarbon fuels in practical premixed combustion devices has been limited by an incomplete understanding of hydrogen-enriched combustion. In particular, the current understanding, in terms of theoretical and computational models, is unable to fully explain the experimental observations for such flames.

A number of previous studies have been conducted on the performance and emission characteristics of practical devices using H2-enriched hydrocarbon fuels [4–7]. It has been found that hydrogen enrichment extends the lean stability limit and decreases emissions of carbon monoxide (CO), nitrogen oxides (NOx), and unburned hydrocarbons (UHC) in spark-ignition engines, power-generation gas turbines, and aircraft gas turbines. There have also been several studies focused on fundamental aspects of hydrogen–hydrocarbon flames. It has been reported that H2-enriched flames display higher laminar flame speeds, extended lean flammability limits, and augmented resistance to strain [8–12]. Moreover, experiments in swirl-stabilized burners [13,14] have been conducted to investigate the stabilization and blowout characteristics of H2-enriched methane–air premixed flames. It has also been shown that the addition of H2 to methane (CH4) extends the flame lean stability limit, allowing stable burner operation at lower flame temperatures and reducing NOx emissions. Of particular note to the present work, Di Sarli and Di Benedetto [15] have identified three regimes of laminar burning velocity as defined by Di Sarli and Di Benedetto [15]. The current understanding, in terms of theoretical and computational models, is unable to fully explain the experimental observations for such flames.

The LES framework developed by Hernández-Pérez et al. [23,24] is used for performing this study. The framework is now briefly summarized, along with the PCM-FPI combustion model.

2.1. Favre-filtered governing equations

For the LES computations considered here, the Favre-filtered form of the Navier–Stokes equations governing compressible flows of a thermally perfect reactive gaseous mixture of N species, neglecting Dufour, Soret and radiation effects, is used to describe the turbulent premixed combustion processes. Relevant flow parameters, \( \phi \), are either filtered using a low-pass spatial filtering procedure or Favre-filtered using a mass-weighted filter to yield \( \bar{\phi} \) or \( \phi \), respectively. The resulting equations are given by

\[
\frac{\partial (\rho \bar{u}_i)}{\partial t} + \frac{\partial (\rho u_i \bar{u}_j)}{\partial x_j} = \frac{\partial \tau_{ij}}{\partial x_j} + \rho \bar{g}_i, \quad (1)
\]

\[
\frac{\partial (\rho \bar{u}_k)}{\partial t} + \frac{\partial (\rho \bar{u}_i \bar{u}_k)}{\partial x_i} + \frac{\partial \rho}{\partial x_i} \bar{u}_k = \frac{\partial \tau_{ik}}{\partial x_k} + \rho \bar{g}_k + A_k, \quad (2)
\]

\[
\frac{\partial (\bar{p} \bar{E})}{\partial t} + \frac{\partial (\bar{p} \bar{E} \bar{u}_i)}{\partial x_i} = \frac{\partial}{\partial x_i} \left( \frac{\partial \bar{E}}{\partial x_i} \bar{u}_i \right) - \frac{\partial \tau_{ik}}{\partial x_k} + \rho g + B_1 + B_2, \quad (3)
\]

\[
\frac{\partial (\bar{p} \tilde{Y}_k)}{\partial t} + \frac{\partial (\bar{p} \tilde{Y}_k \bar{u}_i)}{\partial x_i} + \frac{\partial \tau_{ik}}{\partial x_k} = \tilde{c}_k + C_k, \quad (4)
\]

where \( \bar{p} \) is the filtered mixture density, \( \bar{u}_i \) is the Favre-filtered mixture velocity, \( \bar{E} \) is the Favre-filtered mixture pressure, \( \tilde{Y}_k \) is the Favre-filtered mass fraction of species \( k \), \( \bar{E} \) is the Favre-filtered total mixture energy (including chemical energy) given by \( \bar{E} = \sum_{k=1}^{N} \tilde{Y}_k (\bar{h}_k + \Delta H^0_h) - \bar{p}/\bar{\rho} + \bar{u}_i \bar{u}_i/2 \); \( \bar{h}_k \), \( \Delta H^0_h \) and \( \tilde{c}_k \) are the sensible enthalpy (evaluated in terms of the Favre-filtered temperature), heat of formation and the filtered reaction rate of species \( k \), respectively, and \( g_k \) is the acceleration due to gravity. The filtered equation of state adopts the form \( \bar{p} = \rho \bar{E} \), assuming that the SFS temperature-species correlation is negligible. Here, \( \bar{T} \) is the Favre-filtered mixture temperature and \( \bar{R} \) is the gas constant. The resolved stress tensor, \( \tau_{ik} \), the
resolved total heat flux, \( q_t \), and the resolved species diffusive
fluxes, \( \mathbf{J}_{ix} \), are evaluated in terms of the filtered quantities.

The terms, \( A_1, B_1, B_2, B_3, \) and \( C_1 \), arise from the low-pass
filtering process and require modelling. These terms are expressed as
\[
A_1 = - \frac{\partial}{\partial x_i} \left[ \rho \left( \bar{u}_i \bar{u}_i - \bar{u}_i \bar{u}_i \right) \right]
B_1 = - \frac{\partial}{\partial x_i} \left[ \rho \left( \bar{h}_i \bar{h}_i - \bar{h}_i \bar{h}_i \right) \right]
B_2 = - \frac{1}{2} \frac{\partial}{\partial x_i} \left[ \rho \left( \bar{u}_i \bar{u}_i \bar{u}_i - \bar{u}_i \bar{u}_i \bar{u}_i \right) \right]
C_1 = \frac{\partial}{\partial x_i} \left[ \rho \left( \bar{h}_i \bar{u}_i - \bar{h}_i \bar{u}_i \right) \right]
B_3 = - \frac{1}{2} \frac{\partial}{\partial x_i} \left[ \rho \left( \bar{u}_i \bar{u}_i \bar{u}_i - \bar{u}_i \bar{u}_i \bar{u}_i \right) \right]
\]
and must be modelled for closure of the filtered equation set. The
subfilter stresses, \( \sigma_{ij} = -\rho (\bar{u}_i \bar{u}_j - \bar{u}_i \bar{u}_j) \), are modelled using
an eddy-viscosity type model with \( \sigma_{ij} = 2 \rho \nu (\bar{S}_{ij} - \bar{S}_{ij} / 3) + \delta \rho \phi / 3 \). The SFS turbulent viscosity, \( \nu_s \), is prescribed herein by
using a one-equation model \[25\] for the SFS turbulent kinetic
energy, \( \nu_s \). Standard gradient-based approximations are used in
this work for the modelling of the SFS fluxes \( B_3, B_3, \) and \( C_1 \).

The subfilter turbulent diffusion term, \( B_2 \), is modelled as
suggested by Knight et al. \[26\] with \(-\rho (\bar{u}_i \bar{u}_i \bar{u}_i - \bar{u}_i \bar{u}_i \bar{u}_i) / 2 = \sigma_{ij} \bar{u}_i \).
In the expressions above, Einstein summation convention applies to
the indices \( i, j \) and \( l \).

### 2.2. PCM-FPI combustion model

A primary challenge in the development of LES for turbulent
reactive flows is the accurate and reliable modelling of the
interaction between turbulence and chemistry and the speci-
fication of the filtered reaction rates, \( \bar{\omega}_c \). For LES of
turbulent premixed combustion, this is further complicated by the large
number of species and complexity of the chemical kinetic
mechanisms for typical hydrocarbon fuels and the fact that
the chemical reactions occur in a thin reacting layer at
extremely small scales that are not resolvable on typical LES
grids. This places particular importance on the SFS modelling
of these terms.

The PCM-FPI \[20\] is an approach that combines a presumed
probability density function (PDF) approach with FPI tabulated
chemistry based on prototypical flames with simplified flow
geometry \[21\]. The FPI tabulation technique, which is very
similar to the flamelet-generated manifold (FGM) technique
\[27–29\], helps to greatly reduce the costs of performing reactive
flow computations involving large chemical kinetic
mechanisms and the use of a presumed PDF allows for the
economical treatment of the effects of subfilter turbulence on
the filtered reaction rates.

For the premixed flames of interest here, steady one-
dimensional laminar premixed flamelets are used in the
construction of the FPI tabulated chemistry. Relevant chemical
parameters, \( \phi \), such as species mass fractions and/or re-
action rates, are related to a single progress of reaction
variable, \( Y_c \). For a given equivalence ratio, \( \phi_0 \), the relationship
for \( \phi = \phi^f \phi_0 \) and the progress variable is established by
a mapping of computed solutions for steady-state one-
dimensional laminar premixed flames expressed as a function
of position normal to the flame front on to the \( Y_c \)-space. For
LES of turbulent premixed flames, filtered quantities are then
obtained via
\[
\phi_f = \int_0^1 \phi^f (\bar{c}) \bar{P}(\bar{c}) \, d\bar{c},
\]
where \( \bar{c} \) is the progress variable and \( \bar{P}(\bar{c}) \) is the filtered
probability density function of \( \bar{c} \), which needs to be specified.

The PDF of \( c \) is taken here to be a \( \beta \)-distribution \[30,31\]
and can be constructed from the resolved or filtered progress
variable, \( \bar{c} \), and its SFS variance, \( \sigma_{c,\text{SFS}} \), respectively.
These two variables, \( \bar{c} \) and \( \sigma_{c,\text{SFS}} \), are directly linked to the progress of reaction \( Y_c \)
and its SFS variance, \( \sigma_{c,\text{SFS}} \). The filtered progress variable is
defined as the filtered progress of reaction normalized by its
value at equilibrium: \( \epsilon = Y_c / Y_c^{\text{eq}}(\phi_0) \). The variance of \( c \)
may be obtained from the variance of the progress of reaction,
\( \sigma_{c,\text{SFS}} = Y_c - Y_c^{\text{eq}} \) and expressed as
\( \phi_0 = Y_c / Y_c^{\text{eq}}(\phi_0) \).

Note that other choices for the presumed PDF of \( c \) are possible,
including the original and modified laminar-flame-based
PDFs of Bray et al. \[32\] and Jin et al. \[33\], respectively. While
the correctness of the modified laminar-flame-based
PDF of Jin et al. \[33\] as compared to the \( \beta \)-distribution for pre-
mixed flames has been demonstrated by Salehhi et al. \[34,35\],
for LES of premixed flames in the flamelet regime with relatively
low turbulence intensities, the differences in the predicted
solutions obtained using the modified laminar-flame-based
and \( \beta \) PDFs have been found to be rather small \[34–36\]. For
this reason, it is felt that the \( \beta \)-PDF is sufficiently accurate for the
hydrogen-enriched flames of interest and is used here.

Modelled balance equations are used to determine \( Y_c \) and
\( \phi_0 \). The modelled transport equation for \( Y_c \) has the form
\[
\frac{\partial \bar{Y}_c}{\partial t} + \frac{\partial (\bar{u}_i \bar{Y}_c)}{\partial x_i} = \frac{\partial}{\partial x_i} \left[ D_{Y_i} + D_{Y_i} \right] \frac{\partial \bar{Y}_c}{\partial x_i} + \bar{\omega}_c,
\]
where \( \bar{\omega}_c \) is a source term due to chemistry, \( D_{Y_i} \) is the
diffusion coefficient associated with \( Y_c \), and \( D_{Y_i} \) is the turbulent
diffusion coefficient used to model SFS scalar transport. The
transport equation for \( \phi_0 \) is given by
\[
\frac{\partial \bar{\phi}_0}{\partial t} + \frac{\partial (\bar{u}_i \bar{\phi}_0)}{\partial x_i} = \frac{\partial}{\partial x_i} \left[ D_{\phi_0} + D_{\phi_0} \right] \frac{\partial \bar{\phi}_0}{\partial x_i} + 2 \bar{P} \left( D_{\phi_0} + D_{\phi_0} \right) \frac{\partial \bar{Y}_c}{\partial x_i} \frac{\partial \bar{Y}_c}{\partial x_i} - 2 \bar{P} D_{\phi_0} \frac{\partial \bar{Y}_c}{\partial x_i} \frac{\partial \bar{Y}_c}{\partial x_i}
\]
The scalar dissipation rate of \( Y_c \)
\[
\bar{\omega}_c = 2 \rho D_{\phi_0} \frac{\partial \bar{Y}_c}{\partial x_i} \frac{\partial \bar{Y}_c}{\partial x_i}
\]
may be decomposed into resolved and unresolved parts as follows:
\[
\bar{\omega}_c = 2 \rho D_{\phi_0} \frac{\partial \bar{Y}_c}{\partial x_i} \frac{\partial \bar{Y}_c}{\partial x_i} + 2 \bar{\sigma}_{\phi_0}.
\]

The SFS component is closed with the combined linear
relaxation hypothesis and bimodal limit closure proposed by
Domingo et al. \[30\], which takes into account that \( Y_c \) is a
reactive scalar and its gradient is influenced by chemistry.
It is important to remark that a reaction rate can be written as \( \dot{\omega} = \rho \dot{\omega} \), therefore \( \dot{\omega} = \rho \dot{\omega} \) and \( Y_c \dot{\omega}_c = \rho Y_c \dot{\omega}_c \). The latter is a reaction rate term appearing in the transport equation for \( Y_c \). The terms \( \dot{\omega}_c \) and \( Y_c \dot{\omega}_c \) are included in the tabulated database. By introducing the segregation factor, \( S_c = c_c/(\bar{c}(1 - \bar{c})) \), a look-up table of filtered quantities, \( \bar{c}_c^{\text{PCM}}(\bar{c}_0, \bar{c}, S_c) \), can be pre-generated for use in subsequent computations.

### 2.3. Parallel finite-volume scheme

The Favre-filtered transport equations for a premixed reactive gaseous mixture given by Eqs. (1)–(4) above, along with transport equations for the filtered progress of reaction variable and its variance (Eqs. (6) and (7)) as well as the SFS turbulent kinetic energy, are all solved on multi-block body-fitted meshes consisting of hexahedral cells by employing a second-order accurate parallel finite-volume scheme [23,24,37–42]. The inviscid flux at each cell face is evaluated using limited linear reconstruction [43] and Riemann-solver based flux functions [44,45], while the viscous flux is evaluated using a hybrid average gradient-diamond path method [46]. A standard, explicit, two-stage, second-order-accurate, Runge-Kutta, time-marching scheme is used to integrate forward in time the non-linear, coupled-system of ordinary differential equations resulting from the finite-volume spatial discretization procedure. Parallel implementation of the solution method has been carried out via domain decomposition using the C++ programming language and the MPI (message passing interface) library [47,48]. Second-order accuracy of the finite-volume scheme for discretizations on multi-block body-fitted meshes has been demonstrated in other previous studies [37–40].

### 3. Tabulated chemistry for H₂-enriched methane–air flames

Previous studies by Fiorina et al. [49] and Galpin et al. [31] have shown that for FPI tabulated chemistry of methane–air combustion based on steady laminar premixed flamelets, an appropriate choice for the progress of reaction variable is \( \dot{\omega}_c = Y_c \dot{\omega}_c \). It has also been shown by these authors that it can be sufficient to tabulate the flamelet data for a rather reduced number of species based on their contributions to mixture mass and energy. In fact, just 10 species: CH₄, O₂, N₂, H₂O, CO₂, CO, H₂, H, OH and C₂H₂, are needed to accurately represent the mass and energy of methane–air reactive mixtures [31]. However, in order to improve the prediction of the \( H_2 \) mass fraction and allow for the prediction of NO in H₂-enriched methane–air flames, a revised progress of reaction variable is needed and an enlarged set of species must be tracked. Furthermore, to aid in the treatment of the preferential diffusion of \( H_2 \), transport equations for the tracked species are solved directly, along with the equations associated with the progress variable and its variance, using reconstructed filtered reaction rates from the pre-computed FPI look-up tables based on a high Damkölmer number approximation [30,31].

#### 3.1. Selection of progress variable and reduced set of tracked species

In order to assess the FPI tabulation needs for LES of hydrogen-enriched methane combustion, steady-state, one-dimensional solutions of H₂-enriched methane–air premixed flames were first obtained using the Cantera package [50] with detailed chemistry provided by the GRI-Mech 3.0 chemical kinetics mechanism [51]. GRI-Mech 3.0 has been used previously to obtain good predictions of laminar flame speeds, profiles of major species, NO, levels, and extinction strain rates in agreement with experimental values [10,52]. The generated solutions were confined to the methane-dominated regime and included different levels of enrichment up to a 40% mole fraction of \( H_2 \) in the blended fuel, with increments of 5%. The equivalence ratio was varied from values near the lean-flammability limit up to two.

Careful examination and analysis of the laminar premixed flamelet solutions revealed that a suitable single progress of reaction variable for hydrogen-enriched methane combustion, in the methane-dominated regime, is provided by \( Y_c = Y_{CO} + Y_{NO} \). It was found that this definition of \( Y_c \) increases monotonically and can adequately account for the slow chemistry associated with NO for both lean and stoichiometric H₂-enriched methane–air premixed flamelets. Note that previous studies have shown that NO must be included in the definition of the progress of reaction variable in order to: (i) avoid errors in NO mass fraction that can be introduced when employing a progress variable defined only in terms of major species [53,54]; and (ii) properly account for time scales that differ from those of major species [55,56].

Fig. 1(a) and (b) display the NO mass fraction evolution in the progress variable space (normalized progress of reaction variable) corresponding to the proposed progress of reaction and the progress of reaction variable used previously by Fiorina et al. [49], for lean (\( \varphi = 0.8 \)) and stoichiometric H₂-enriched methane–air premixed flamelets. Magnified views of the regions where NO is reaching its equilibrium value are also displayed in Fig. 1(c) and (d). The profiles of NO mass fraction corresponding to \( Y_c \) based on major species show larger slopes when NO is near its equilibrium value, which are almost vertical. This may lead to errors in the tabulated NO mass fraction. On the other hand, the NO profiles corresponding to the new definition of \( Y_c \) show smoother and monotone transitions to the equilibrium values, as can be seen in Fig. 1(c) and (d).

Temperature and representative species mass fractions profiles in \( Y_c \)-space, associated with the proposed progress of reaction variable, are also depicted in Fig. 2 for premixed flamelets enriched with \( H_2 \) by 20% (on a molar basis) and a range of equivalence ratios that includes fuel-rich conditions. It is apparent from the results shown in the figure that the novel \( Y_c \) is suitable for lean and stoichiometric conditions, ensuring a one-to-one correspondence between the representative mass fractions and \( Y_c \). However, for the fuel-rich conditions (see Fig. 2(b) for \( \varphi = 1.5 \) and Fig. 2(c) for \( \varphi = 1.25 \) and \( \varphi = 1.5 \)), these desirable properties would no longer hold for the proposed progress of reaction variable. Nevertheless, since the scope of this research work is limited to lean flames, the newly defined \( Y_c \) is adopted herein.
Furthermore, based on contributions to mixture mass, energy, and heat release, the species CH$_4$, C$_2$H$_2$, OH, O$_2$, O, N$_2$, H$_2$O, CO$_2$, CO, H$_2$, H, and OH were selected, in addition to NO. Two further species, C$_2$H$_2$ and NH$_3$, were added to allow for elemental mass balance and absorb inconsistencies for the C and H elements. As a result, the number of species included in the FPI look-up tables for the LES of hydrogen-enriched methane-flames considered here was 14 species.

Fig. 1 – NO mass fraction evolution in the progress variable space for $Y_\text{C} = Y_{\text{CO}_2} + Y_{\text{CO}}$ and $Y_\text{N} = Y_{\text{NO}}$, corresponding to lean and stoichiometric conditions ($\phi = 0.8$ and $\phi = 1.0$) with different levels of H$_2$ (0%, 20%, and 40% on a molar basis) in the blended H$_2$–CH$_4$ fuel. One-dimensional steady-state laminar premixed flames. Top: full NO evolution in progress variable space. Bottom: magnified views of the regions where NO is reaching its equilibrium value.

Fig. 2 – Temperature and representative species profiles in the progress of reaction space for $Y_\text{C} = Y_{\text{CO}_2} + Y_{\text{NO}}$, corresponding to lean, stoichiometric, and rich conditions, and 20% H$_2$ in the blended H$_2$–CH$_4$ fuel. One-dimensional steady-state laminar premixed flames.
3.2. Performance of proposed tabulation procedure for laminar premixed flames

The proposed 14-species FPI tabulation method with $Y_c = Y_{CO2} + Y_{NO}$ was found to yield accurate solutions for steady, one-dimensional, laminar, premixed $H_2$-$CH_4$-$air$ flames in the methane-dominated regime to within a few percent of the values of the full chemical kinetic solutions, for thermodynamic properties and major and minor species. For example, consider the comparison of FPI tabulated results obtained using the approach outlined above to detailed chemical kinetic solutions obtained using Cantera given in Figs. 3 and 4 for one-dimensional premixed, laminar methane-$air$ and hydrogen-enriched methane-$air$ (29% $H_2$ on a molar basis) flames at atmospheric pressure and having a fresh gas temperature of 300 K. The equivalence ratio for both cases is $\phi = 0.52$. The predicted temperature and CO and NO mass fraction profiles of the one-dimensional flames are displayed in the two figures. Two sets of results for the PCM-FPI approach are given, one based on reading mass fractions directly and the other based on transporting the species with reconstructed reaction rates. The predicted profiles of the species CO and NO closely follow the profiles given by the Cantera solutions. In particular, there is a remarkable agreement for the equilibrium NO levels, which are less than one part per million (ppm) in each case. The PCM-FPI predictions of the laminar flame speed and burned gas temperature for these flames also compare well with those given by Cantera. A summary of flame speed and temperature comparisons are provided in Table 1. Note that the PCM-FPI solutions based on reconstructed reaction rates provide slight improvements in the predictions, particularly for flame temperature.

3.3. Generation of PCM-FPI tables

Based on the proposed 14-species set and progress of reaction variable, $Y_c$, defined by $Y_c = Y_{CO2} + Y_{NO}$, PCM-FPI look-up tables for hydrogen-enriched methane-$air$ combustion were generated here by using pre-computed laminar premixed flamelet solutions obtained using Cantera and GRI-Mech 3.0.
The flamelet solutions were integrated with \( \beta \)-PDFs for a range of values of both the resolved and SFS variance of the progress variable. Values for the filtered species mass fractions and filtered reaction rates \( \tilde{\omega}_Y \) and \( \tilde{\gamma}_C \) were pre-computed and stored for retrieval from the look-up tables, which contained 138 non-uniformly-spaced values of \( \tilde{c} \) and 25 non-uniformly-spaced values of \( S_c \).

### 4. LES results for Bunsen-type premixed flames

LES computations for axisymmetric Bunsen-type turbulent premixed methane–air and \( \text{H}_2 \)-enriched methane–air flames have been performed. The predicted solutions for both flames are examined and further compared to available data from the experiments carried out earlier by Halter et al. [22].

#### 4.1. Burner setup

The Bunsen-type flames of interest correspond to lean premixed turbulent conditions at an equivalence ratio of \( \phi = 0.6 \) and atmospheric pressure [22]. For the enriched flame, \( \text{H}_2 \) represents 20% of the blended fuel on a molar basis (\( \chi = 0.2 \)). An axisymmetric burner with an inner nozzle diameter of 25 mm was used to generate the premixed turbulent conical flames stabilized by annular pilot flames. The turbulence at the burner exit was characterized by a turbulence intensity \( u' = 0.17 \text{ m/s} \) and an integral length scale \( L = 3 \text{ mm} \). The mixtures of reactants had a temperature of 300 K and their mean inflow velocity was 2.1 m/s.

In the simulations, a cylindrical domain having a diameter of 0.05 m and a height of 0.1 m was employed and discretized with a grid consisting of 1,638,400 hexahedral cells. The cylindrical computational domain and mesh are shown in Fig. 5. The pilot flames were approximated by a uniform inflow of hot combustion products at a velocity of 2.73 m/s. For the burner exit, a uniform mean inflow of reactants with superimposed turbulent fluctuations was prescribed. The same turbulence field was used for both the enriched and pure methane flame simulations. The assumed Schmidt numbers associated with each species, as taken from the Cantera flamelet solutions and used for the Bunsen flame simulations, are given in Table 2.

#### 4.2. Instantaneous flame fronts

Three-dimensional views of the predicted instantaneous flame surfaces for both the pure methane and hydrogen-enriched flames, identified by the isotherm \( T = 650 \text{ K} \), are depicted in Fig. 6 corresponding to a physical time \( t = 56 \text{ ms} \), for which a quasi-steady flame structure has been achieved in each case. The isotherms (light grey) are shown interacting with turbulent structures identified by iso-values of the Q-criterion, \( Q = 1606 \text{ s}^{-2} \) (cyan). The Q-criterion [57] can be used to visualize the coherent structures of the turbulent velocity field. Positive values of Q are associated with regions where vorticity dominates straining.

From Fig. 6 it is evident that the simulated flames exhibit a wrinkled surface having a similar structure up to 5 cm above the burner exit. Further downstream, the hydrogen enriched

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**Table 1** — Summary of predicted flame speeds and temperatures in the burned gas for one-dimensional steady-state \( \text{CH}_4 \)-air \((\chi = 0.0) \) and \( \text{H}_2 \)-\( \text{CH}_4 \)-air \((\chi = 0.29) \) premixed flames at \( \phi = 0.52 \). PCM-FPI model with 14 species reading mass fractions directly (FPI-SP) and transporting the species using reconstructed reaction rates (FPI-RR).

<table>
<thead>
<tr>
<th></th>
<th>Cantera</th>
<th>PCM-FPI</th>
<th>FPI-SP</th>
<th>FPI-RR</th>
</tr>
</thead>
<tbody>
<tr>
<td>( x )</td>
<td>0.0</td>
<td>0.29</td>
<td>0.0</td>
<td>0.29</td>
</tr>
<tr>
<td>( s_L ) (m/s)</td>
<td>0.0612</td>
<td>0.0806</td>
<td>0.061</td>
<td>0.081</td>
</tr>
<tr>
<td>( T_b ) (K)</td>
<td>1510.99</td>
<td>1523.89</td>
<td>1511.02</td>
<td>1526.9</td>
</tr>
</tbody>
</table>

---

**Table 2** — Schmidt numbers of the species for the lean \( \text{CH}_4 \)-air \((\chi = 0.0) \) and \( \text{H}_2 \)-\( \text{CH}_4 \)-air \((\chi = 0.2) \) premixed flames at \( \phi = 0.6 \), taken from the Cantera solutions in the burned gas.

<table>
<thead>
<tr>
<th>Species</th>
<th>( \beta )-PDF</th>
<th>Cantera</th>
<th>PCM-FPI</th>
</tr>
</thead>
<tbody>
<tr>
<td>( \text{H}_2 )-( \text{CH}_4 )-air flame ((20% \text{H}_2))</td>
<td></td>
<td></td>
<td></td>
</tr>
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<tr>
<td>( \text{NH}_3 )</td>
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**Fig. 5** — Cylindrical computational domain and mesh containing 1,638,400 cells used in the LES predictions of the turbulent premixed Bunsen-type flames.
flame displays a higher degree of wrinkling with sharper ridges of negative curvature (curvature is negative in regions that are concave with respect to the unburned gas) and larger pockets of positive curvature, which are more pronounced near the flame tip. The vortical structures identified by the Q-criterion are similar in shape and number near the burner exit for both flames. Further downstream, the number of the coherent vortices decreases more rapidly for the pure methane–air flame, whereas a large number of turbulent structures is observed for the H₂-enriched flame, particularly in the highly wrinkled region near the flame tip. The addition of H₂ would seem to modify the interaction between the flame front and the turbulent field, which can be related to the less diffusive-thermally stable character of the H₂-enriched flame. It has been suggested that the diffusive-thermal instability could lead to self-turbulization of flames [58].

More details of the internal structure of the flames can be seen in Fig. 7, where planar cuts of the instantaneous mass fraction distributions of representative species are respectively shown. It is evident that the enriched flame is shorter, which can be ascribed to a faster consumption of the blended fuel. The enriched flame was found to have slightly higher flow temperatures, particularly in regions convex towards the combustible mixture of gases. In addition, more and sharper cusps that protrude the burned gas are observed in the enriched flame. Reduced levels of CO₂ (not shown) were also observed in the computed enriched flame. In contrast, the planar cuts of NO mass fractions of Fig. 7 display larger amounts of NO in the enriched flame. Areas of larger NO concentrations are localized downstream regions of positive curvature (convex towards the unburned gas) for the enriched case. These regions also coincide with areas of high temperature. The planar cuts of CO mass fractions show an intensified production of CO for the enriched flame in the highly wrinkled region near the flame tip, around the centerline. However, no significant difference of CO levels is noticeable in the fully burned gas zones of the two flames from these contours.

Fig. 6 – Instantaneous iso-surface $T=650$ K of the Bunsen-type flames (light grey) interacting with vortical structures (cyan) identified by $Q = 0.5(\bar{u}/L)^2 = 1606$ s$^{-2}$ at 56 ms. Left: CH₄–air flame ($\alpha=0$). Right: H₂–CH₄–air flame ($\alpha=0.2$).

Fig. 7 – Planar cuts of the instantaneous CO and NO mass fraction distributions of the Bunsen-type flames at 56 ms. Plane $y = 0$. Left: CH₄–air flame ($\alpha=0$). Right: H₂–CH₄–air flame ($\alpha=0.2$).
4.3. Average flame structure

To compare the numerical results with the experimental data of Halter et al. [22], a time-average of the LES solutions was performed over 70 ms (from 30 ms to 100 ms) using 71 instantaneous snapshots of each solution. Moreover, 2D slices of the resolved LES temperature field were extracted and processed to calculate progress variable fields based on temperature. The experimental flame brush was calculated using 500 Mie scattering images of oil droplets seeded in the fresh mixture of reactants [22]. The images were converted to a binary format and an edge-finding algorithm was applied to each instantaneous image to determine the progress variable maps. The resolution of the Mie scattering imaging procedure was 110 μm/pixel.

The experimental and numerical flame brushes are given in Fig. 8. Although clear differences between the experimental and numerical results can be seen, both the experiments and the simulations show a decrease in the flame height (36% and 18%, respectively, computed from the temperature-based progress variable $c_T = 0.1$ contour and evaluated at the centerline [22]) when hydrogen is added to methane, indicating a higher global burning rate for the enriched flame. It can also be observed that the predicted flames are taller than their experimental counterparts. It is worth noting that pockets of unburned gas were neglected (i.e., they were treated as burned gas) in the image analysis of Halter [22]. The presence of pockets of unburned gas near and above the flame tip in the instantaneous images would increase the height of the ensemble-averaged flame front. Therefore, the actual flame brushes are expected to be somewhat taller than those shown in Fig. 8.

Two-dimensional curvature was also extracted from instantaneous unfiltered experimental images and slices of the numerical solutions. The curvature PDFs from the experimental data and the LES solutions associated to the $c_T = 0.5$ contour are shown in Fig. 9. The PDFs display a Gaussian-type shape centred around zero. In both the experiments and the simulations, the addition of hydrogen lead to slightly broader PDFs. Furthermore, the hydrogen enriched flame displayed a higher degree of wrinkling with sharper ridges of negative curvature.

5. Conclusions

In summary, the LES simulations of axisymmetric Bunsen-type turbulent premixed methane–air and H$_2$–CH$_4$–air flames predicted similar qualitative trends to those found in the experiments of Halter et al. [22] for flame height and curvature. The enriched flame was shorter, which is attributed to a faster consumption of the blended fuel. The curvature PDFs displayed a Gaussian-type shape centred around zero. In both the experiments and the simulations, the addition of hydrogen leads to slightly broader PDFs, in both the experiments and the simulations, with higher probabilities of finding larger curvatures for the enriched flame. This indicates that more small-scale wrinkling is present in the enriched flame.
larger pockets of positive curvature, which were more pronounced near the flame tip. This behaviour has been associated with preferential diffusion effects and the less diffusive—thermally stable character of the H₂-enriched flame. In addition, reduced levels of CO₂ and increased levels of NO emissions were predicted for the enriched flame, and a slight increase in the CO levels in areas of fully burned gas was exhibited in the predicted enriched flame. All in all, the proposed treatment for the PCM-FPI combustion model, in terms of progress variable and tabulated data, appears to perform well for enriched methane flames in the methane-dominated regime. Further study is warranted for a wider class of flames.

Acknowledgements

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